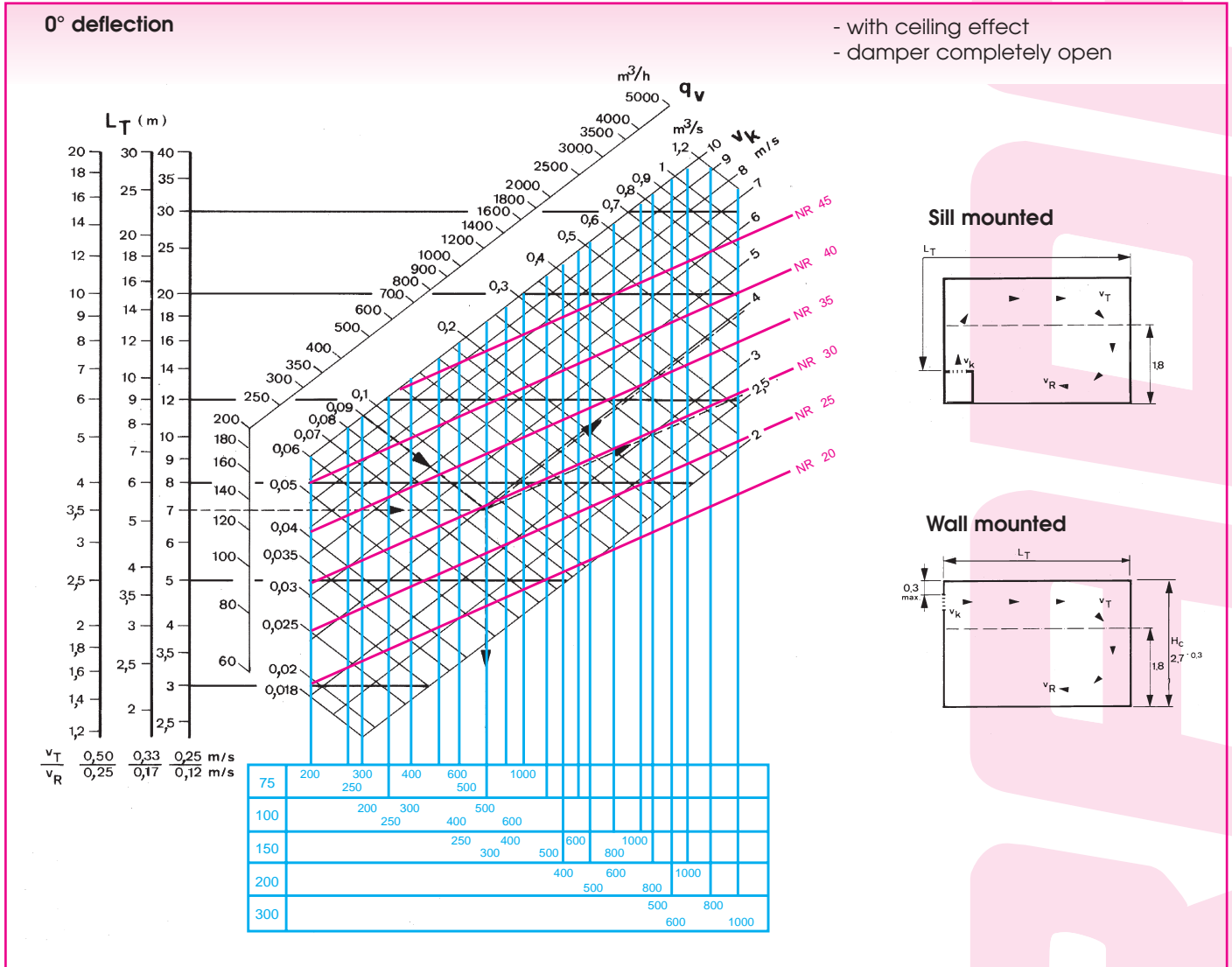
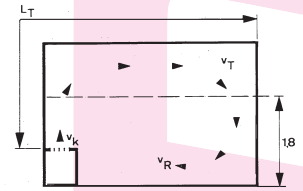


**BAR GRILLE  
A-300/400/500**

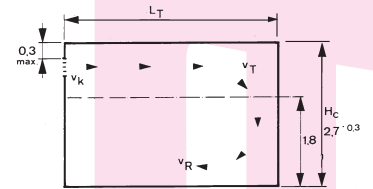
**Selection diagram - supply**



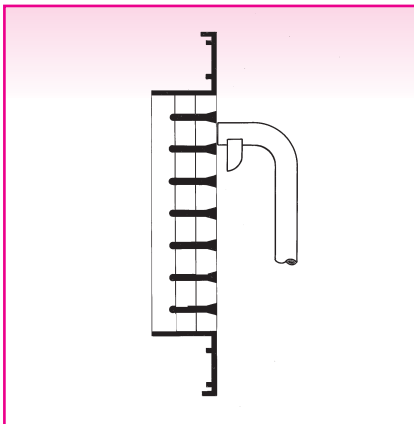
Sill mounted



Wall mounted



**Air flow rate measurement-  
supply**



Velometer jet 2220 A or 6070

A <sub>k</sub> -values (m²)								
H (mm)	L (mm)							
	200	250	300	400	500	600	800	1000
75	0,006	0,008	0,009	0,013	0,016	0,019	0,027	0,031
100	0,009	0,011	0,013	0,019	0,023	0,027	0,038	0,047
150	—	0,019	0,023	0,031	0,038	0,047	0,063	0,078
200	—	—	—	0,042	0,053	0,063	0,084	0,108
300	—	—	—	—	0,084	0,099	0,133	0,167

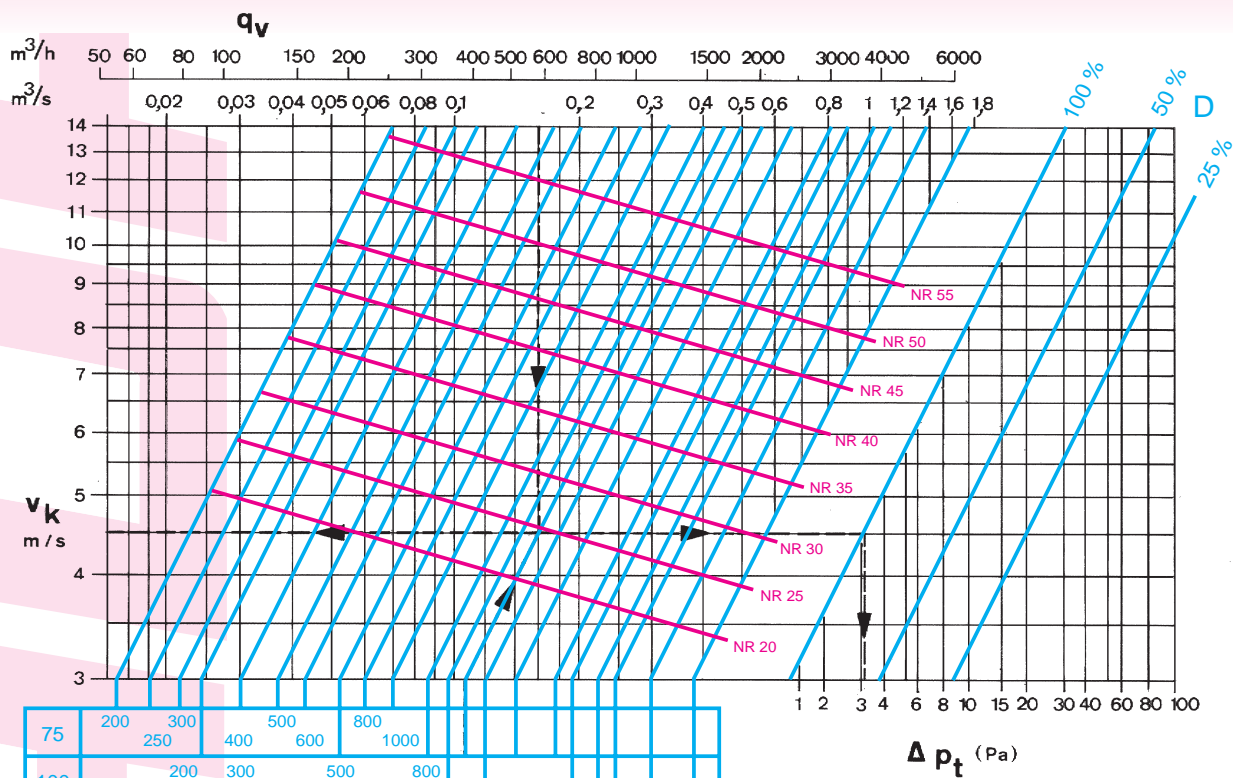
**Correction factors:**

- Throw correction factor without ceiling effect

Distance between ceiling and supply grille	Correction
≥ 0,9 m	L <sub>T</sub> X 0,75

- Correction factors for vertical vane deflection of flow equalizer (see p. 1 231 verso)

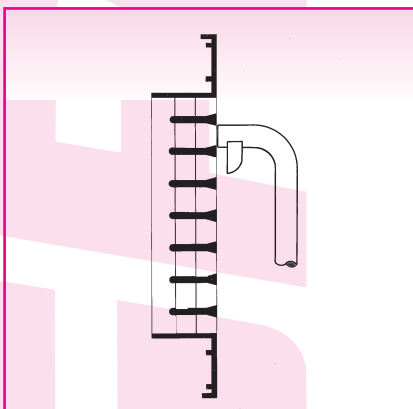
## Selection diagram - exhaust



75	200	300	400	500	600	800	1000		
100		200	300	400	500	600	800		
150			250	400	500	600	800		
200				300	400	500	600	800	
300					400	500	600	800	1000

When 15° deflected bars are used, air flow rate will be reduced by 5% at listed  $\Delta p_t$  and NR values.

## Air flow rate measurement - exhaust

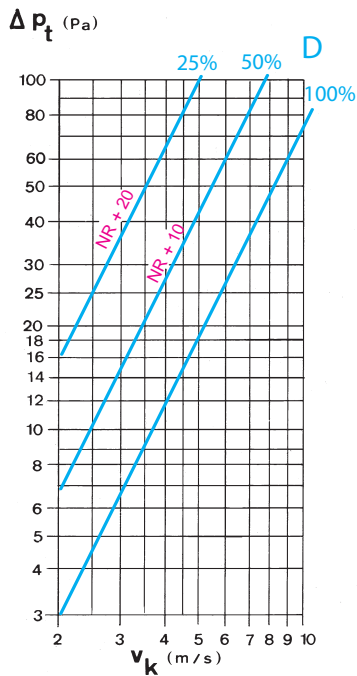


Velometer jet 2220 A or 6070

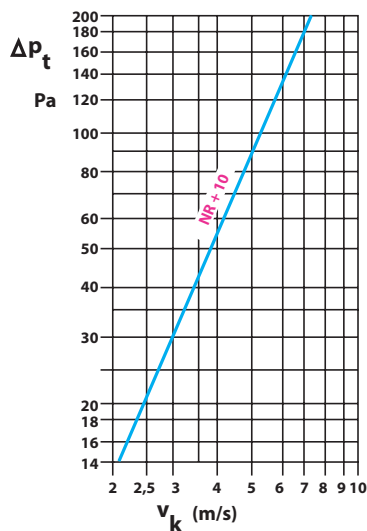
H (mm)	$A_k$ -values ( $m^2$ )							
	L (mm)							
	200	250	300	400	500	600	800	1000
75	0,005	0,006	0,007	0,010	0,012	0,014	0,020	0,023
100	0,007	0,008	0,008	0,014	0,017	0,020	0,028	0,035
150	—	0,014	0,017	0,023	0,028	0,035	0,047	0,058
200	—	—	—	0,031	0,039	0,047	0,063	0,080
300	—	—	—	—	0,063	0,074	0,099	0,125

### Pressure loss - supply

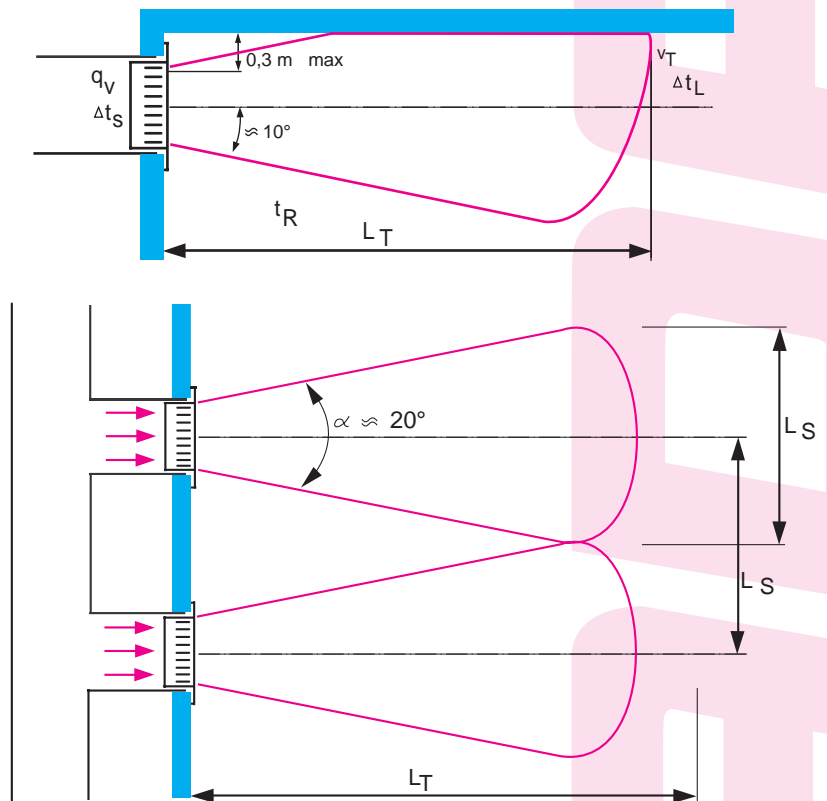
with damper type .. 7



with perforated sheet type .. 3



### Example



#### SUPPLY

##### Selection data:

Air flow rate  $q_v = 0,09 \text{ m}^3/\text{s}$   
 Throw  $L_T = 7 \text{ m}$  at  $v_T = 0,25 \text{ m/s}$ .

##### Solution:

Grille  $500 \times 100$  or  $300 \times 150 \text{ mm}$ .  
 Supply air velocity  $v_k = 3,9 \text{ m/s}$ .  
 Noise level NR 29  
 Total pressure loss with perforated sheet:  $\Delta p_t = 59 \text{ Pa}$ .  
 Noise level correction NR  
 $29 + 10 = \text{NR } 39$

#### EXHAUST

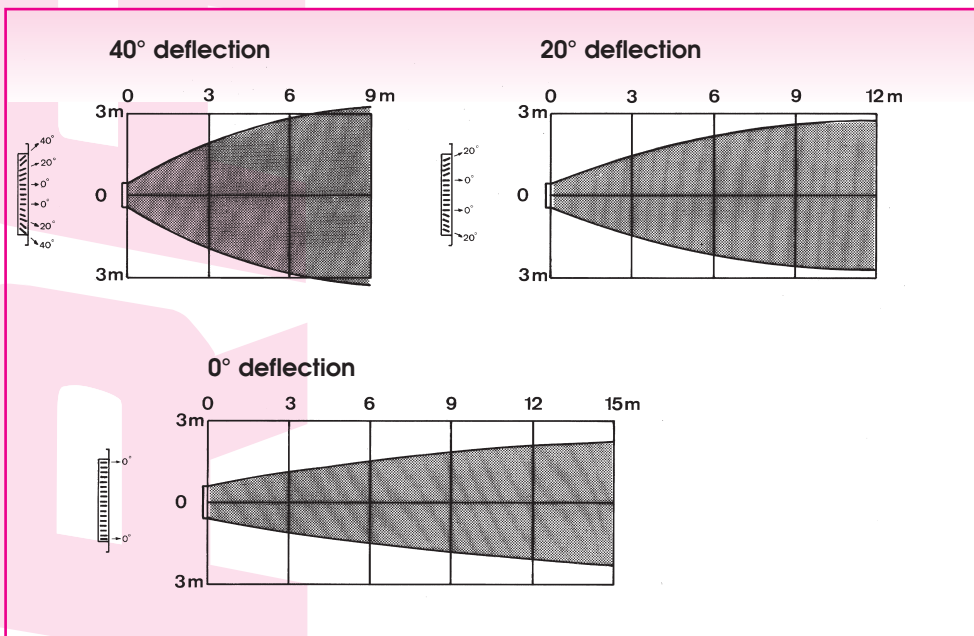
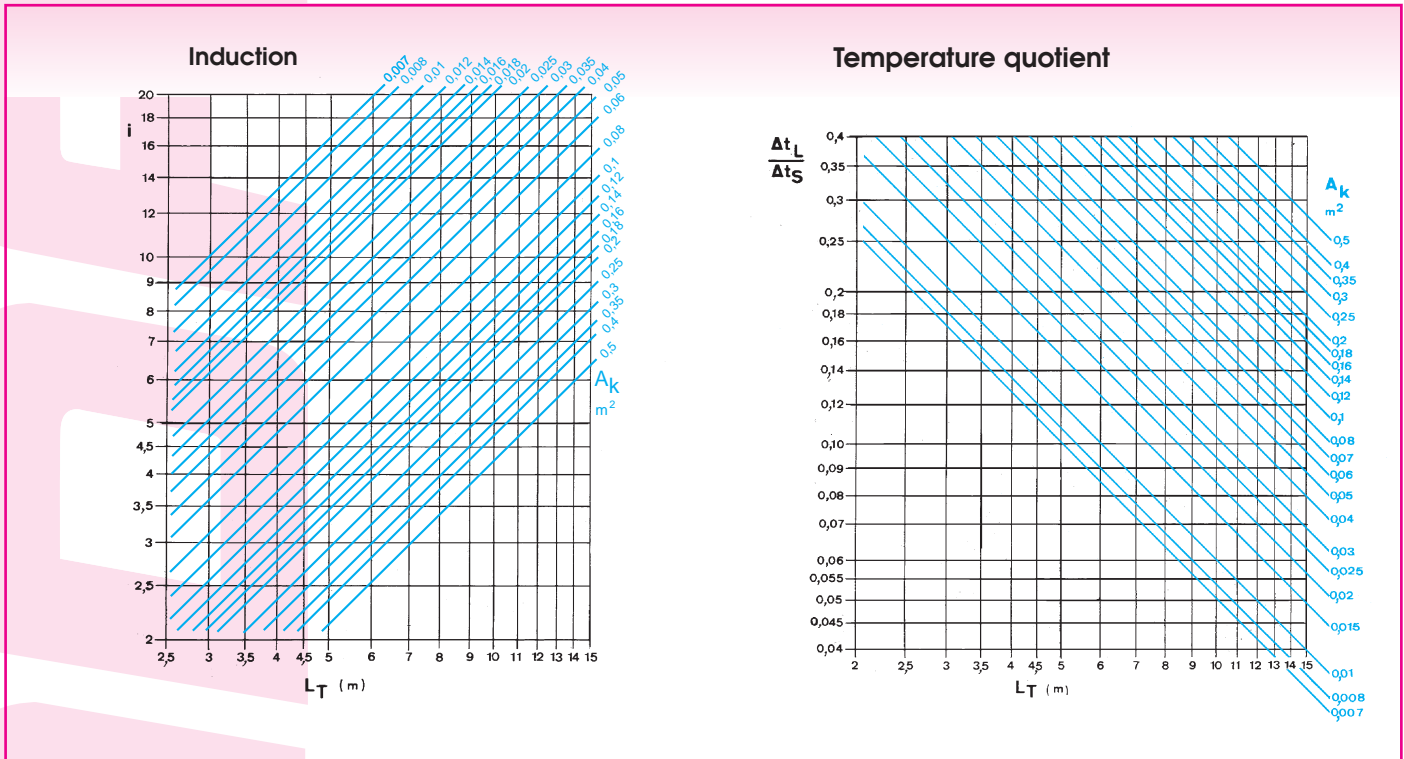
##### Selection data:

Exhaust air flow rate  $q_v = 0,16 \text{ m}^3/\text{s}$

##### Solution:

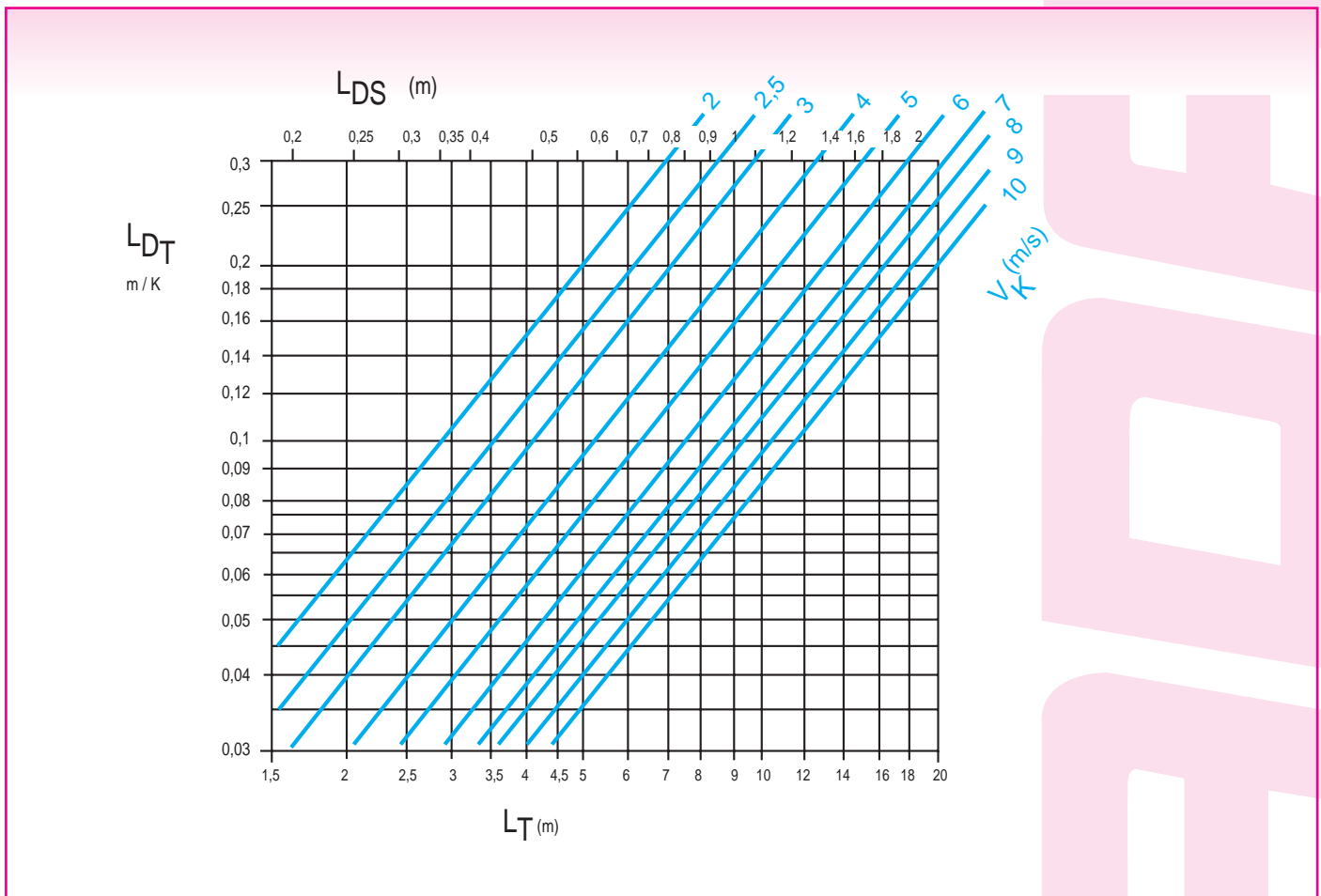
Grille  $1000 \times 100 \text{ mm}$ .  
 Air velocity  $v_k = 3,9 \text{ m/s}$ .  
 Noise level NR 25  
 Total pressure loss with damper  
 100 % open:  $\Delta p_t = 3,2 \text{ Pa}$

**Induction and temperature quotient with ceiling effect (also valid for linear grilles)**



Correction factors Correction factors for vertical vane deflection of flow equalizer	Type	Deflection	$A_k$	$v_k$	$L_T$	NR	$i$	$\frac{\Delta t_L}{\Delta t_S}$
		300, 400	20°	x 0,87	x 1,15	x 0,85	+ 3	x 1,4
		40°	x 0,80	x 1,25	x 0,75	+ 5	x 2	x 0,5

**Drop requirements**

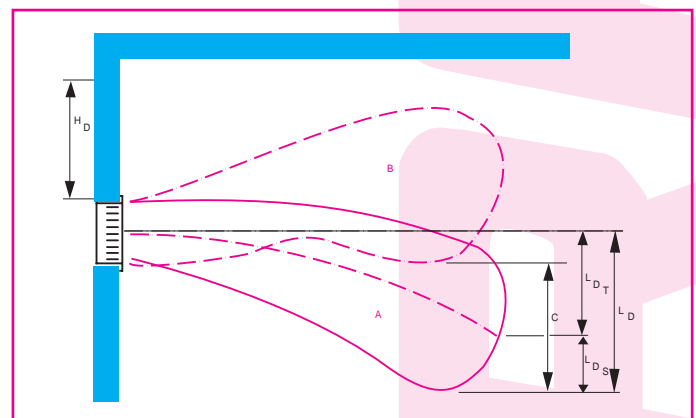


**Drop requirements**

The total drop is the maximum vertical distance between the centre of a grille core and the lower point of a specified envelope, determined by the envelope velocity  $v_T$ .

The total drop consists of two elements:  $L_D = L_{DS} + L_{DT}$

- 1) The isothermal drop  $L_{DS}$  is the distance between the centre of an air current and the lowest point of the envelope.
- 2) The non-isothermal drop  $L_{DT}$  is the distance between the centre of the grille core and the air current centre line, at the place of measurement.

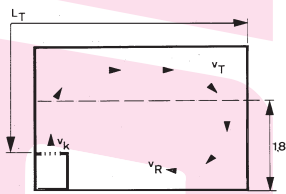


**Selection diagram - supply**

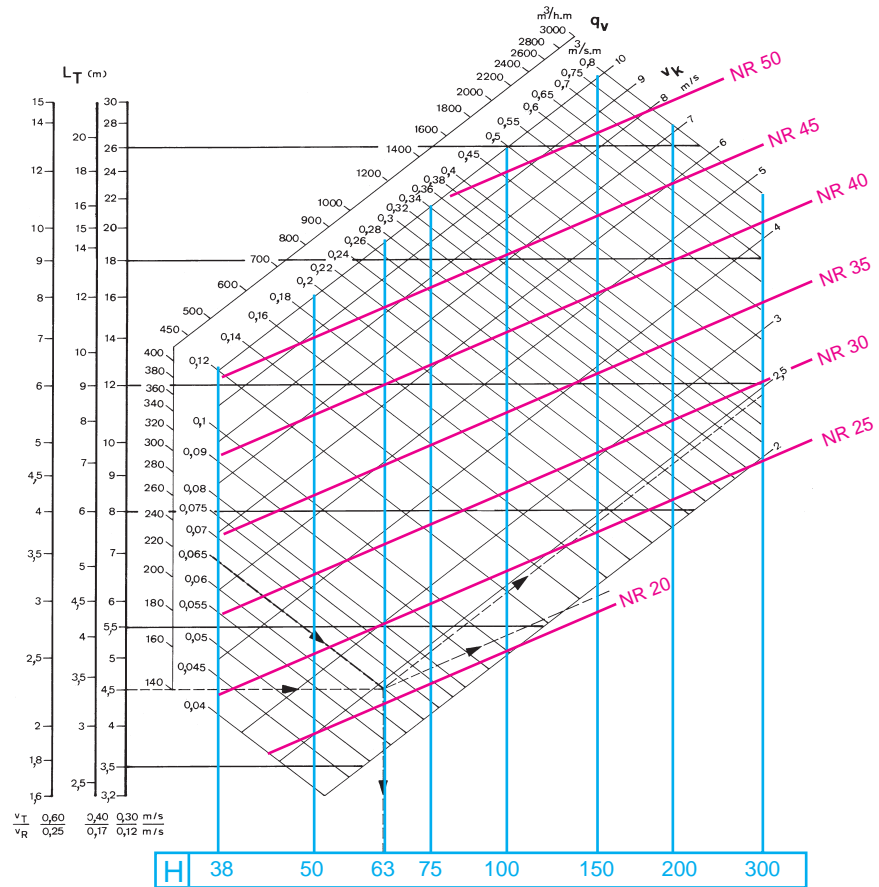
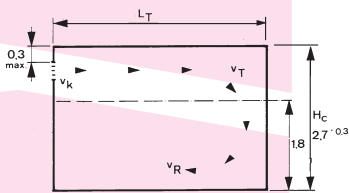
**0° deflection**

- with ceiling effect
- damper completely open

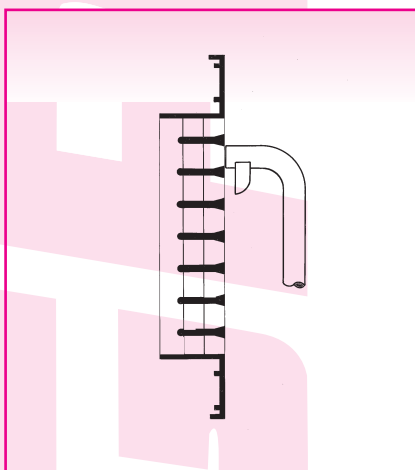
**Sill mounted**



**Wall mounted**



**Air flow rate measurement - supply**

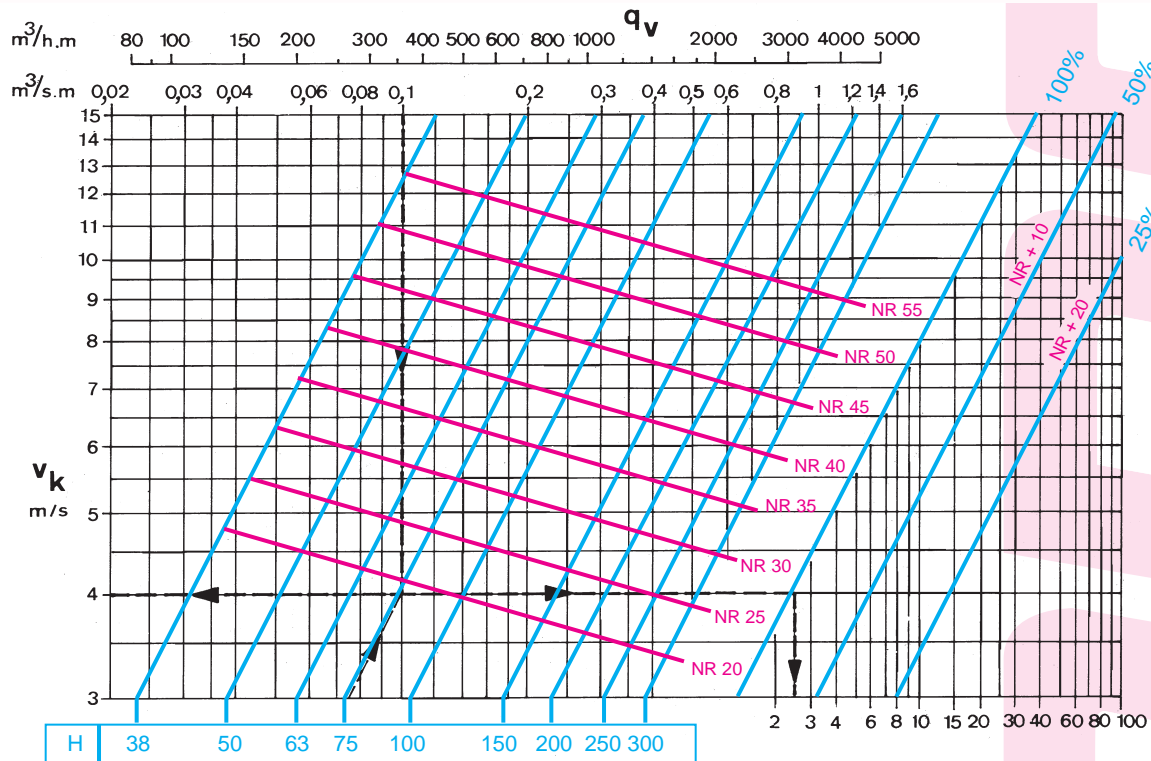


Velometer jet 2220 A or 6070

<b><math>A_k</math>-values (<math>m^2/m</math>)</b>			
<b>H (mm)</b>	<b><math>A_k</math></b>	<b>H (mm)</b>	<b><math>A_k</math></b>
<b>38*</b>	0,012	<b>100</b>	0,049
<b>50*</b>	0,019	<b>150</b>	0,079
<b>63*</b>	0,027	<b>200</b>	0,110
<b>75</b>	0,034	<b>300</b>	0,171

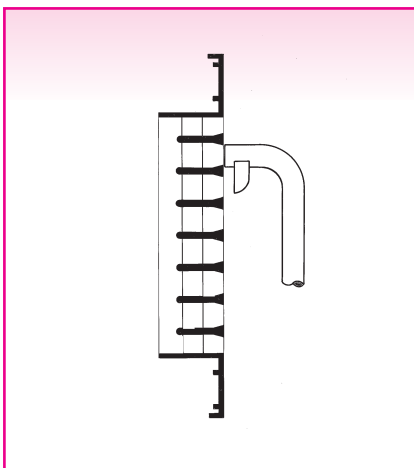
\* A-400/500 only

**Selection diagram - exhaust**



When 15° deflected bars are used, air flow rate will be reduced by 5% at listed  $\Delta p_t$  and NR values.

**Air flow rate measurement-exhaust**



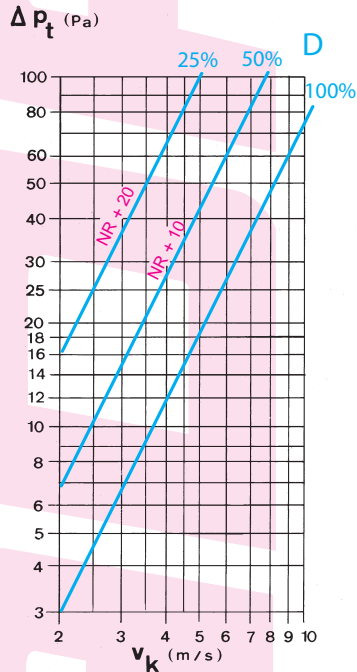
Velometer jet 2220 A or 6070

A <sub>k</sub> -values (m <sup>2</sup> /m)			
H (mm)	A <sub>k</sub>	H (mm)	A <sub>k</sub>
38*	0,008	125	0,048
50*	0,013	150	0,059
63*	0,019	200	0,082
75	0,025	250	0,105
100	0,036	300	0,127

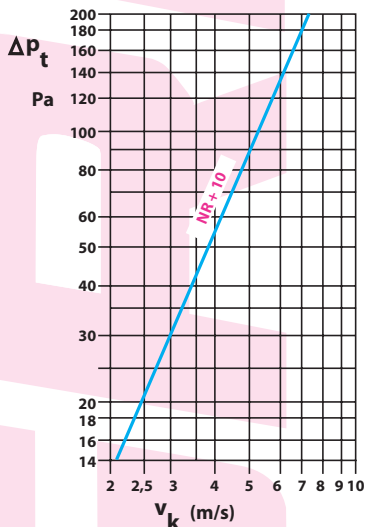
\* A-400/500 only

**Pressure loss - supply**

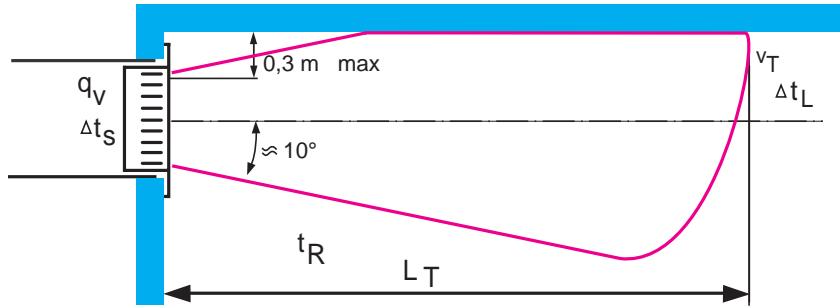
with damper type .. 7



with perforated sheet type .. 3



**Example**



**Correction factors - supply:**

- Throw correction factor without ceiling effect

Distance between ceiling and supply grilles	Correction
≥ 0,9 m	$L_T \times 0,75$

- Correction factors for linear grilles

Grill length (m)	Throw correction	Noise level correction (NR)
1 - 2	$L_T \times 1,00$	+ 0 NR
2 - 6,5	$L_T \times 1,10$	+ 5 NR

**SUPPLY:**

**Selection data:**

Air flow rate  $q_v = 0,065 \text{ m}^3/\text{s}$   
 Throw  $L_T = 4,5 \text{ m}$  at  $v_T = 0,30 \text{ m/s}$

**Solution:**

Size H = 63 mm  
 Supply air velocity  $v_k = 2,4 \text{ m/s}$   
 Noise level NR 21  
 Total pressure with damper 100 % open:  $\Delta p_t = 10 \text{ Pa}$   
 Correction on noise level  
 NR 21 + 10 = NR 31

**EXHAUST:**

**Selection data:**

Air flow rate  $q_v = 0,1 \text{ m}^3/\text{s.m}$

**Solution:**

Size H = 75 mm  
 Air velocity  $v_k = 4 \text{ m/s}$   
 Noise level NR 20  
 Total pressure loss with damper 100 % open:  $\Delta p_t = 2,5 \text{ Pa}$